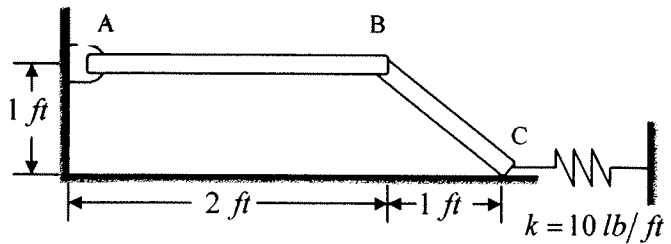


Ex. Bar AB weighs 10 lb and bar BC weighs 6 lb. If the system is released from rest in the position shown, what are the angular velocities of the bars at the instant just before joint B hits the smooth floor?



Known:

$$m_{AB} = 0.311 \frac{\text{lb s}^2}{\text{ft}} \quad m_{BC} = 0.186 \frac{\text{lb s}^2}{\text{ft}} \quad I_{G,AB} = 0.104 \text{ lb s}^2 \text{ ft} \quad I_{G,BC} = 0.031 \text{ lb s}^2 \text{ ft}$$

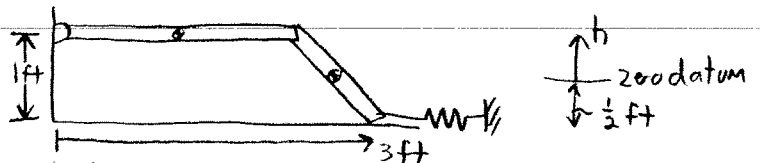
Approach: CoE from release ① to just before impact ② system = AB + BC + spring

$$E_{\text{sys},2} - E_{\text{sys},1} = W_{\text{ext}}$$

$W_{\text{ext}} = 0$ only forces are pin reactions (assume no friction)

$$E_{\text{sys},1} = KE_1 + PE_1 + SE_1 + U_1$$

$\hookrightarrow 0$ no impact
 $\hookrightarrow 0$ uncompressed



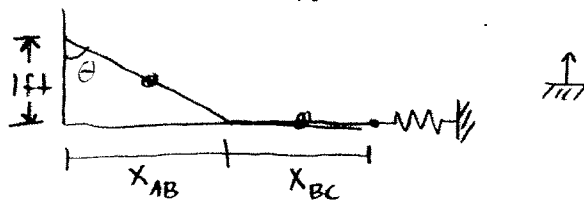
$$PE_1 = m_{AB}g \left(\frac{1}{2} \text{ ft}\right) + m_{BC}g(\phi)$$

\hookrightarrow at datum

$KE_1 = 0$ starting from rest

$$E_{\text{sys},2} = KE_2 + PE_2 + SE_2 + U_2$$

$\hookrightarrow 0$



$$X_{AB}^2 = 2^2 - 1^2 \quad \therefore X_{AB} = \sqrt{3 \text{ ft}^2} = 1.732 \text{ ft}$$

$$X_{BC}^2 = 1^2 + 1^2 \quad \therefore X_{BC} = \sqrt{2 \text{ ft}^2} = 1.414 \text{ ft}$$

$$X_{AB} + X_{BC} = 3.146 \text{ ft} \quad \theta = 60^\circ$$

$$SE_2 = \frac{1}{2} k \Delta x^2 = \left(\frac{1}{2}\right) \left(10 \frac{\text{lb}}{\text{ft}}\right) (3.146 \text{ ft} - 3 \text{ ft})^2 = 0.107 \text{ lb ft}$$

$$PE_2 = m_{BC}g \left(-\frac{1}{2} \text{ft}\right)$$

$$KE_2 = \frac{1}{2} m_{AB} v_{GAB}^2 + \frac{1}{2} m_{BC} v_{GBC}^2 + \frac{1}{2} I_{GAB} \omega_{AB}^2 + \frac{1}{2} I_{GBC} \omega_{BC}^2$$

Together, CoE gives

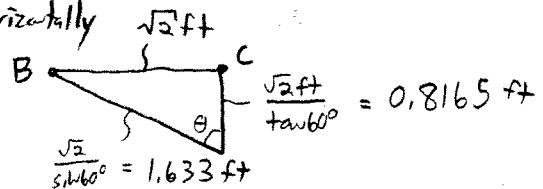
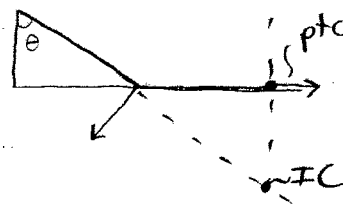
$$\textcircled{1} \quad \frac{1}{2} [m_{AB} v_{GAB}^2 + m_{BC} v_{GBC}^2 + I_{GAB} \omega_{AB}^2 + I_{GBC} \omega_{BC}^2] - m_{BC}g \left(\frac{1}{2} \text{ft}\right) + 0.10716 \text{ft} - m_{AB}g \left(\frac{1}{2} \text{ft}\right) = 0$$

Kinematics of bar AB - fixed pivot at pt A

$$\text{use } v = \omega r \text{ to get } v_{GAB} = \omega_{AB} (1 \text{ft}) \quad \textcircled{2}$$

$$v_B = \omega_{AB} (2 \text{ft}) \quad \textcircled{3}$$

Kinematics of bar BC - GPM use IC



$$\textcircled{4} \quad v_B = \omega_{BC} (1.633 \text{ft})$$

$$\textcircled{5} \quad v_{GBC} = \omega_{BC} (1.08 \text{ft})$$



$$r_{GBC/IC}^2 = \left(\frac{\sqrt{2}}{2}\right)^2 + (0.8165)^2$$

$$r_{GBC/IC} = 1.08 \text{ft}$$

UNKNOWN: $v_{GAB}, v_{GBC}, \omega_{AB}, \omega_{BC}, v_B$

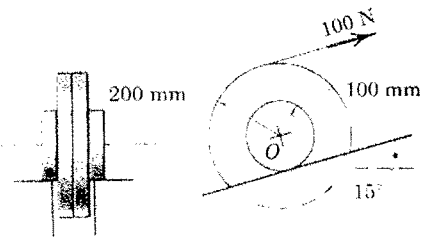
$$\omega_{AB} = 4.479 \text{ rad/s } (-R)$$

$$\omega_{BC} = 5.487 \text{ rad/s } (R)$$

Solving using vector approach (without IC) is much more complex

Ex. 6/9 The wheel rolls up the incline on its hubs without slipping and is pulled by the 100-N force applied to the cord wrapped around its outer rim. If the wheel start from rest, compute its angular velocity after its center has moved a distance of 3-m up the incline. The wheel has a mass of 40-kg with a center of mass at O and has a centroidal radius of gyration of 150-mm. (taken from Engineering Mechanics, 4th Edition by Meriam & Kraige)

$$I_G = 0.9 \text{ kg m}^2$$



Approach: CoE (two points in space) from start ① to 3m up slope ②

COE set PE datum to give zero PE at time ①, system = wheel only

$$E_{\text{sys},1} = KE_1 + PE_1 + SE_1 + U_1$$

$\begin{matrix} \rightarrow 0 & \rightarrow 0 \\ \text{by our defn of PE datum} \\ \rightarrow 0 \text{ starts from rest} \end{matrix}$

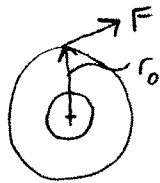
$$E_{\text{sys},2} = KE_2 + PE_2 + SE_2 + U_2$$

$$PE_2 = mgh \text{ with } h = (3\text{m})\sin 15^\circ$$



$$KE_2 = \frac{1}{2} m v_G^2 + \frac{1}{2} I_G \omega^2$$

$$W_{\text{ext}} = \int_1^2 \vec{F} \cdot d\vec{s} = (\text{force})(\text{distance}) = (100\text{N})(6\text{m})$$



$$\begin{aligned} \text{distance} &= (\# \text{ revs}) (\text{circumference at cord}) \\ &= \left(\frac{3\text{m}}{2\pi(0.1\text{m})} \right) (2\pi r_0) \end{aligned}$$

$$= \left(\frac{30}{2\pi} [\text{rad}] \right) (2\pi(0.2\text{m})) = 6\text{m}$$

Together $E_{\text{sys},2} - E_{\text{sys},1} = W_{\text{ext}}$

$$mg(3\text{m}\sin 15^\circ) + \frac{1}{2} m v_G^2 + \frac{1}{2} I_G \omega^2 - 0 = (100\text{N})(6\text{m}) \quad (1)$$

Kinematics

using contact point as IC and $\vec{v}_G = \vec{\omega} \times \vec{r}_{G/IC}$

$$\vec{v}_G = (-\omega R) \times (0.1\text{m})(-\sin 15^\circ \hat{i} + \cos 15^\circ \hat{j})$$

$$v_{Gx} = (\omega)(0.1\text{m})(\cos 15^\circ)$$

$$v_{Gy} = (\omega)(0.1\text{m})(\sin 15^\circ)$$

$$\left. \begin{matrix} v_{Gx} = (\omega)(0.1\text{m})(\cos 15^\circ) \\ v_{Gy} = (\omega)(0.1\text{m})(\sin 15^\circ) \end{matrix} \right\} v_G = (\omega)(0.1\text{m}) \quad (2)$$

gets $\vec{\omega} = 21.33 \text{ rad/s } (-\hat{k})$
 $v = 2.13 \text{ m/s}$