

Fig. P15.59 and P15.60

Find: ω_{AD}
 \vec{v}_B
 \vec{v}_A

No masses or forces given,
 use kinematics!



15.60 Knowing that at the instant shown the velocity of collar D is 1.8 m/s upward, determine (a) the angular velocity of rod AD, (b) the velocity of point B, (c) the velocity of point A.

Given $\vec{v}_D = 1.8 \text{ m/s } \hat{j}$

Constraint on pt B $\vec{v}_B = v_B \hat{x}$ (rigid rod EB prevents vertical motion)

Relate $\vec{v}_D + \vec{v}_B$: $\vec{v}_B = \vec{v}_D + \vec{\omega}_{AD} \times \vec{r}_{B/D}$

$\vec{r}_{B/D} = (0.3 \text{ m}) (-\cos 30^\circ \hat{x} + \sin 30^\circ \hat{j})$
 $= -0.2598 \hat{x} + 0.15 \hat{j} \text{ m}$
 $\vec{\omega}_{AD} = \omega_{AD} \hat{k}$

$v_B \hat{x} = 1.8 \text{ m/s } \hat{j} + (\omega_{AD} \hat{k}) \times (-0.2598 \hat{x} + 0.15 \hat{j} \text{ m})$

$v_B \hat{x} = 1.8 \text{ m/s } \hat{j} - 0.2598 \text{ m } \omega_{AD} \hat{j} - 0.15 \text{ m } \omega_{AD} \hat{x}$

\hat{x}) $v_B = (-0.15 \text{ m}) \omega_{AD}$

\hat{j}) $0 = 1.8 \text{ m/s} - (0.2598 \text{ m}) \omega_{AD}$

from \hat{j} eqn so $\omega_{AD} = 6.928 \text{ rad/s}$ i.e. $\vec{\omega}_{AD} = 6.928 \hat{k} \frac{\text{rad}}{\text{s}}$

Plug ω_{AD} into \hat{x} eqn

$v_B = (-0.15 \text{ m})(6.928 \text{ rad/s}) = -1.039 \text{ m/s}$

$\vec{v}_B = -1.039 \text{ m/s } \hat{x}$

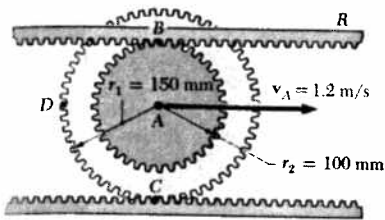
To get pt A: $\vec{v}_A = \vec{v}_D + \vec{\omega}_{AD} \times \vec{r}_{A/D}$

$\vec{r}_{A/D} = (0.5 \text{ m}) (-\cos 30^\circ \hat{x} + \sin 30^\circ \hat{j})$

$\vec{v}_A = (1.8 \text{ m/s}) \hat{j} + (6.928 \hat{k} \frac{\text{rad}}{\text{s}}) \times (0.5 \text{ m}) (-\cos 30^\circ \hat{x} + \sin 30^\circ \hat{j})$

$= (1.8 \text{ m/s} - 3 \text{ m/s}) \hat{j} + (-1.732 \text{ m/s}) \hat{x}$

$\vec{v}_A = -1.732 \text{ m/s } \hat{x} - 1.2 \text{ m/s } \hat{j}$ or $v_A = 2.11 \text{ m/s}$ $\nabla 34.7^\circ$

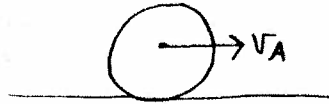


SAMPLE PROBLEM 15.2

The double gear shown rolls on the stationary lower rack; the velocity of its center A is 1.2 m/s directed to the right. Determine (a) the angular velocity of the gear, (b) the velocities of the upper rack R and of point D of the gear.

a) No masses, no forces \Rightarrow try kinematics

Consider the rolling on stationary lower track



The gear teeth enforce no slipping, IC is contact pt

$$\vec{v}_A = \vec{\omega} \times \vec{r}_{A/IC} \quad w/ \quad \vec{r}_{A/IC} = r_1 \hat{j}$$

$$\vec{\omega} = \omega \hat{k}$$

$$1.2 \text{ m/s } \hat{i} = (\omega \hat{k}) \times (r_1 \hat{j})$$

$$1) \quad 1.2 \text{ m/s} = -\omega r_1 \quad \therefore \quad \omega = \frac{-1.2 \text{ m/s}}{0.15 \text{ m}} = -8 \frac{\text{rad}}{\text{sec}} \quad \text{e.g.} \quad \boxed{\vec{\omega} = -8 \frac{\text{rad}}{\text{s}} \hat{k}}$$

b) To find velocity of upper rack, find velocity of pt B

$$\text{IC at contact point: } \vec{v}_B = \vec{\omega} \times \vec{r}_{B/IC} \quad w/ \quad \vec{r}_{B/IC} = (r_1 + r_2) \hat{j}$$

$$= (-8 \frac{\text{rad}}{\text{s}} \hat{k}) \times (r_1 + r_2) \hat{j}$$

$$= (8 \frac{\text{rad}}{\text{s}}) (0.15 \text{ m} + 0.1 \text{ m}) \hat{i}$$

$$\vec{v}_B = 2 \text{ m/s } \hat{i}$$

Point B is in contact with the upper rack, so

$$\boxed{\vec{v}_{\text{upper rack}} = 2 \text{ m/s } \hat{i}}$$

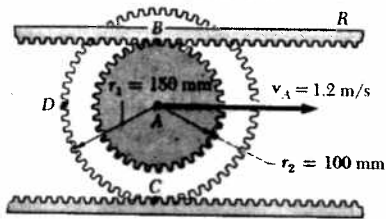
To find velocity of pt D, use IC at contact pt again

$$\vec{v}_D = \vec{\omega} \times \vec{r}_{D/IC} \quad w/ \quad \vec{r}_{D/IC} = (-r_1) \hat{i} + (r_1) \hat{j}$$

$$= (-8 \frac{\text{rad}}{\text{s}} \hat{k}) \times (-r_1 \hat{i} + r_1 \hat{j})$$

$$= (8 \frac{\text{rad}}{\text{s}}) (r_1) \hat{j} + (-8 \frac{\text{rad}}{\text{s}}) (r_1) (-\hat{i})$$

$$\boxed{\vec{v}_D = 1.2 \text{ m/s } \hat{j} + 1.2 \text{ m/s } \hat{i}}$$

**SAMPLE PROBLEM 15.2**

The double gear shown rolls on the stationary lower rack; the velocity of its center A is 1.2 m/s directed to the right. Determine (a) the angular velocity of the gear, (b) the velocities of the upper rack R and of point D of the gear.

a) no masses, no forces \Rightarrow try kinematics

The gear travels on the lower rack as it turns:

$$\text{distance traveled} = (\text{rotation angle}) \left(\frac{1 \text{ rev}}{2\pi \text{ rad}} \right) \left(\frac{2\pi r_1 [\text{m}]}{1 \text{ rev}} \right)$$

$$x_A = \frac{2\pi r_1 \theta}{2\pi} = r_1 \theta$$

$$\text{diff wrt time} \quad v_A = r_1 \dot{\theta} = r_1 \omega$$

$$\omega = \frac{v_A}{r_1} = \frac{(1.2 \text{ m/s})}{(0.15 \text{ m})} = 8 \frac{\text{rad}}{\text{sec}}$$

note that to move gear to right, $\vec{\omega}$ must be cw, so

$$\vec{\omega} = -8 \frac{\text{rad}}{\text{s}} \hat{k}$$

$$\begin{aligned} \vec{v}_B &= \vec{v}_A + \vec{\omega} \times \vec{r}_{B/A} \quad \text{w/} \quad \vec{r}_{B/A} = r_2 \hat{j} \\ &= (1.2 \text{ m/s} \hat{i}) + (-8 \frac{\text{rad}}{\text{s}} \hat{k}) \times (r_2 \hat{j}) \\ &= 1.2 \text{ m/s} \hat{i} + 0.8 \text{ m/s} \hat{i} = 2 \text{ m/s} \hat{i} \end{aligned}$$

$$\vec{v}_R = 2 \text{ m/s} \hat{i}$$

$$\begin{aligned} \vec{v}_D &= \vec{v}_A + \vec{\omega} \times \vec{r}_{D/A} \quad \text{w/} \quad \vec{r}_{D/A} = (-r_1 \hat{i}) \\ &= (1.2 \text{ m/s} \hat{i}) + (-8 \frac{\text{rad}}{\text{s}} \hat{k}) \times (-r_1 \hat{i}) \\ &= 1.2 \text{ m/s} \hat{i} + 1.2 \text{ m/s} \hat{j} \end{aligned}$$

$$\vec{v}_D = 1.2 \hat{i} + 1.2 \hat{j} \text{ m/s}$$