## ECE-320: Linear Control Systems Homework 4

Due: Tuesday March 29 at the beginning of class **Exam #1**, Thursday March 31

1) (sisotool problem) For the plant modeled by the transfer function

$$G_1(s) = \frac{6000}{s^2 + 4s + 400}$$

You are to design a PI controller, a PID controller with complex conjugate zeros, and a PID controller with real zeros that meet the following specifications

$$PO \le 10\%$$
$$T_s \le 2.5 \text{ sec}$$
$$k_p \le 0.5$$
$$k_i \le 5$$
$$k_d \le 0.01$$

In *sisotool*, in the LTI viewer, if you right click on the graph and select **Characteristics** you can let *sisotool* find the settling time. You should copy your step response and root locus plots to a word document, as well as including your values of the controller coefficients.

2) (sisotool problem) For the plant modeled by the transfer function

$$G_2(s) = \frac{6250}{s^2 + 0.5s + 625}$$

You are to design a PI controller, a PID controller with complex conjugate zeros, and a PID controller with real zeros that meet the following specifications

$$PO \le 10\%$$

$$PI, T_s \le 15.0 \text{ sec}, PID T_s \le 0.5 \text{ sec}$$

$$k_p \le 0.5$$

$$k_i \le 5$$

$$k_d \le 0.01$$

In *sisotool*, in the LTI viewer, if you right click on the graph and select **Characteristics** you can let *sisotool* find the settling time. You should copy your step response and root locus plots to a word document, as well as including your values of the controller coefficients.

**3**) One of the things that will be coming up in lab more and more is the limitation of the amplitude of the control signal, or the *control effort*. This is also a problem for most practical systems. In this problem we will do some simple analysis to better understand why Matlab's sisotool won't give us a good estimate of the control effort for some types of systems, and why dynamic prefilters can often really help us out here.

a) For the system below,

$$R(s) \xrightarrow{F} G_{pf}(s) \xrightarrow{+} G_{c}(s) \xrightarrow{U(s)} G_{p}(s) \xrightarrow{+} G_{c}(s) \xrightarrow{U(s)} G_{p}(s) \xrightarrow{+} G_{c}(s) \xrightarrow{U(s)} G_{c}(s) \xrightarrow{+} G_{c}(s) \xrightarrow{U(s)} G_{c}(s) \xrightarrow{+} G_{c}(s) \xrightarrow{+} G_{c}(s) \xrightarrow{U(s)} G_{c}(s) \xrightarrow{+} G_{c}(s) \xrightarrow{+} G_{c}(s) \xrightarrow{U(s)} G_{c}(s) \xrightarrow{+} G_{c}(s) \xrightarrow{+}$$

show that U(s) and R(s) are related by

$$U(s) = \frac{G_c(s)G_{pf}(s)}{1 + G_p(s)G_c(s)}R(s)$$

b) For many types of controllers, the maximum value of the control signal is just after the step is applied, at  $t = 0^+$ . Although most of the time we are concerned with steady state values and use the final value Theorem in the *s*-plane, in this case we want to use the initial value Theorem, which can be written as

$$\lim_{t\to 0^+} u(t) = \lim_{s\to\infty} sU(s)$$

If the system input is a step of amplitude A, show that

$$u(0^+) = \lim_{s \to \infty} \frac{AG_c(s)G_{pf}(s)}{1 + G_p(s)G_c(s)}$$

This result shows very clearly that the initial control signal is directly proportional to the amplitude of the input signal, which is pretty intuitive.

c) Now let's assume

$$G_{c}(s) = \frac{N_{c}(s)}{D_{c}(s)} \quad G_{p}(s) = \frac{N_{p}(s)}{D_{p}(s)} \quad G_{pf}(s) = \frac{N_{pf}(s)}{D_{pf}(s)}$$

If we want to look at the initial value for a unit step, we need to look at

$$u(0^{+}) = \lim_{s \to \infty} \frac{sG_{c}(s)G_{pf}(s)}{1 + G_{c}(s)G_{p}(s)} \frac{1}{s} = \lim_{s \to \infty} \frac{G_{c}(s)G_{pf}(s)}{1 + G_{c}(s)G_{p}(s)}$$

Let's also then define

$$\tilde{U}(s) = \frac{G_c(s)G_{pf}(s)}{1 + G_c(s)G_p(s)}$$

so that

$$u(0^+) = \lim_{s \to \infty} \tilde{U}(s)$$

Show that

$$\tilde{U}(s) = \frac{N_{pf}(s)}{\left(\frac{D_{c}(s)}{N_{c}(s)}\right)D_{pf}(s) + \left(\frac{N_{p}(s)}{D_{p}(s)}\right)D_{pf}(s)}$$

and

$$\deg \tilde{U} = \deg N_{pf} - \max \left[ \deg D_c - \deg N_c + \deg D_{pf}, \deg N_p - \deg D_p + \deg D_{pf} \right]$$

where  $\deg Y$  is the degree of polynomial Y.

d) Since we are going to take the limit as  $s \to \infty$ , we need the degree of  $\tilde{U}(s)$  to be less than or equal to zero for a step input to have a finite  $u(0^+)$ . Why?

For our 1 dof systems in lab, we have  $\deg N_p = 0$  and  $\deg D_p = 2$ . Use this for the remainder of this problem

e) If the prefilter is a constant, show that in order to have a finite  $u(0^+)$  we must have

$$\deg D_c \ge \deg N_c$$

f) If the numerator of the prefilter is a constant, then in order to have a finite  $u(0^+)$  we must have

$$\deg D_c - \deg N_c + \deg D_{pf} \ge 0 \text{ or } -2 + \deg D_{pf} \ge 0$$

g) For P, I, D, PI, PD, PID, and lead controllers, determine if  $u(0^+)$  is finite if the prefilter is a constant.

Note: Although it may appear that the control effort is sometimes infinite, in practice this is not true since our motor cannot produce an infinite signal. This large initial control signal is referred to as a *setpoint kick*. There are different ways to implement a PID controller to avoid this, and we will cover two of them in Lab 4.