Name: $\qquad$
Circle One: $\quad$ Richards - $03 \quad$ Richards - 04

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Problem 1 ( 32 )
Problem 2 (34) $\qquad$
Problem 3
( 34 ) $\qquad$

TOTAL
( 100 ) $\qquad$

## General Comments

(1) Anytime you apply conservation or accounting principles in solving a problem, sketch and clearly identify the system you have selected. In addition, clearly indicate how your assumptions or given information simplifies the general equations. Numbered-symbols, e.g. $P_{1}$, in equations must make sense for the problem in question, i.e. they must match you figure and communicate information accurately.
(2) Guidelines - Open text book / closed notes; however, you may use the following
... your equation page (single side of $8-1 / 2 \times 11$ sheet of paper)
... unit conversion page provide with text
... your computer to run MAPLE assuming it is not communicating with any networks and you are not using any previously prepared worksheets, etc. Use of EES is forbidden.
(3) For maximum credit,
... solve problems symbolically first showing logic and reasoning for solution,
... substitute numbers into the equations clearly showing any required unit conversion factors
... then and only then crunch numbers on your calculator.
If I only have to punch your numbers into a calculator to get a correct answer (including units) you will receive full credit. Don't make me guess what you are doing.
(4) Watch the time and feel free to remove the staple and take the test apart so that you don't have to keep flipping pages around.
PLEASE REMOVE THE STAPLE AND USE EXTRA PAPER INSTEAD OF WRITING ON THE BACK OF PAGES AND THEN HAVING TO FLIP BACK AND FORTH TO FINISH THE PROBLEM. I'LL GLADLY RESTAPLE YOUR TEST! If you remove the staple, PLEASE place your name on every page you turn in.

## Length

$$
\begin{aligned}
& 1 \mathrm{ft}=12 \mathrm{in}=0.3048 \mathrm{~m}=1 / 3 \mathrm{yd} \\
& 1 \mathrm{~m}=100 \mathrm{~cm}=1000 \mathrm{~mm}=39.37 \mathrm{in}=3.2808 \mathrm{ft} \\
& 1 \mathrm{mile}=5280 \mathrm{ft}=1609.3 \mathrm{~m}
\end{aligned}
$$

## Mass

$1 \mathrm{~kg}=1000 \mathrm{~g}=2.2046 \mathrm{lbm}$
$1 \mathrm{lbm}=16 \mathrm{oz}=0.45359 \mathrm{~kg}$
1 slug = 32.174 lbm

## Temperature Values

$(\mathrm{T} / \mathrm{K})=\left(\mathrm{T} /{ }^{\circ} \mathrm{R}\right) / 1.8$
$(\mathrm{T} / \mathrm{K})=\left(\mathrm{T} /{ }^{\circ} \mathrm{C}\right)+273.15$
$\left(\mathrm{T} /{ }^{\mathrm{O}} \mathrm{C}\right)=\left[\left(\mathrm{T} /{ }^{\circ} \mathrm{F}\right)-32\right] / 1.8$
$\left(\mathrm{T} /{ }^{\circ} \mathrm{R}\right)=1.8(\mathrm{~T} / \mathrm{K})$
$\left(\mathrm{T} /{ }^{\circ} \mathrm{R}\right)=\left(\mathrm{T} /{ }^{\circ} \mathrm{F}\right)+459.67$
$\left(\mathrm{T} /{ }^{\circ} \mathrm{F}\right)=1.8\left(\mathrm{~T} /{ }^{\circ} \mathrm{C}\right)+32$

## Temperature Differences

$\left(\Delta \mathrm{T} /{ }^{\circ} \mathrm{R}\right)=1.8(\Delta \mathrm{~T} / \mathrm{K})$
$\left(\Delta \mathrm{T} /{ }^{\circ} \mathrm{R}\right)=\left(\Delta \mathrm{T} /{ }^{\circ} \mathrm{F}\right)$
$(\Delta \mathrm{T} / \mathrm{K})=\left(\Delta \mathrm{T} /{ }^{\circ} \mathrm{C}\right)$

## Volume

$$
\begin{aligned}
& 1 \mathrm{~m}^{3}=1000 \mathrm{~L}=10^{6} \mathrm{~cm}^{3}=10^{6} \mathrm{~mL}=35.315 \mathrm{ft}^{3}=264.17 \mathrm{gal} \\
& 1 \mathrm{ft}^{3}=1728 \mathrm{in}^{3}=7.4805 \mathrm{gal}=0.028317 \mathrm{~m}^{3} \\
& 1 \mathrm{gal}=0.13368 \mathrm{ft}^{3}=0.0037854 \mathrm{~m}^{3}
\end{aligned}
$$

## Volumetric Flow Rate

$$
\begin{aligned}
& 1 \mathrm{~m}^{3} / \mathrm{s}=35.315 \mathrm{ft}^{3} / \mathrm{s}=264.17 \mathrm{gal} / \mathrm{s} \\
& 1 \mathrm{ft}^{3} / \mathrm{s}=1.6990 \mathrm{~m}^{3} / \mathrm{min}=7.4805 \mathrm{gal} / \mathrm{s}=448.83 \mathrm{gal} / \mathrm{min}
\end{aligned}
$$

## Force

$1 \mathrm{~N}=1 \mathrm{~kg} \cdot \mathrm{~m} / \mathrm{s}^{2}=0.22481 \mathrm{lbf}$
$1 \mathrm{lbf}=1 \mathrm{slug} \cdot \mathrm{ft} / \mathrm{s}^{2}=32.174 \mathrm{lbm} \cdot \mathrm{ft} / \mathrm{s}^{2}=4.4482 \mathrm{~N}$

## Pressure

$1 \mathrm{~atm}=101.325 \mathrm{kPa}=1.01325 \mathrm{bar}=14.696 \mathrm{lbf} / \mathrm{in}^{2}$
$1 \mathrm{bar}=100 \mathrm{kPa}=10^{5} \mathrm{~Pa}$
$1 \mathrm{~Pa}=1 \mathrm{~N} / \mathrm{m}^{2}=10^{-3} \mathrm{kPa}$
$1 \mathrm{lbf} / \mathrm{in}^{2}=6.8947 \mathrm{kPa}=6894.7 \mathrm{~N} / \mathrm{m}^{2}$
[lbf/in ${ }^{2}$ often abbreviated as "psi" ]

## Energy

$1 \mathrm{~J}=1 \mathrm{~N} \cdot \mathrm{~m}$
$1 \mathrm{~kJ}=1000 \mathrm{~J}=737.56 \mathrm{ft} \cdot \mathrm{lbf}=0.94782 \mathrm{Btu}$
$1 \mathrm{Btu}=1.0551 \mathrm{~kJ}=778.17 \mathrm{ft} \cdot \mathrm{lbf}$
$1 \mathrm{ft} \cdot \mathrm{lbf}=1.3558 \mathrm{~J}$

## Energy Transfer Rate

$1 \mathrm{~kW}=1 \mathrm{~kJ} / \mathrm{s}=737.56 \mathrm{ft} \cdot \mathrm{lbf} / \mathrm{s}=1.3410 \mathrm{hp}=0.94782 \mathrm{Btu} / \mathrm{s}$
$1 \mathrm{Btu} / \mathrm{s}=1.0551 \mathrm{~kW}=1.4149 \mathrm{hp}=778.17 \mathrm{ft} \cdot \mathrm{lbf} / \mathrm{s}$
$1 \mathrm{hp}=550 \mathrm{ft} \cdot \mathrm{lbf} / \mathrm{s}=0.74571 \mathrm{~kW}=0.70679 \mathrm{Btu} / \mathrm{s}$

## Specific Energy

$1 \mathrm{~kJ} / \mathrm{kg}=1000 \mathrm{~m}^{2} / \mathrm{s}^{2}$
$1 \mathrm{Btu} / \mathrm{lbm}=25037 \mathrm{ft}^{2} / \mathrm{s}^{2}$
$1 \mathrm{ft} \cdot \mathrm{lbf} / \mathrm{lbm}=32.174 \mathrm{ft}^{2} / \mathrm{s}^{2}$

Problem 1 (32 points)
Complete the information in the table for refrigerant 134a (R-134a). Entries that require calculations are worth 4 points. Entries that only require a simple look up are worth 2 points.

For phase, indicate whether it is a compressed liquid (CL), saturated liquid (SL), saturated mixture (SM), saturated vapor (SV), or superheated vapor (SHV).

For properties and quality, provide a number or indicate it is not applicable (N/A).

| State | Phase | $\boldsymbol{T}$ <br> ${ }^{\circ} \mathrm{C}$ | $\boldsymbol{P}$ <br> MPa | $\boldsymbol{x}$ | $\boldsymbol{u}$ <br> $\mathrm{m}^{3} / \mathrm{kg}$ | $\boldsymbol{h}$ <br> $\mathrm{kJ} / \mathrm{kg}$ | $\boldsymbol{s}$ <br> $\mathrm{kJ} /(\mathrm{kg} \cdot \mathrm{K})$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 |  | 60 |  |  |  | 172.71 |  |
| 2 |  | 60 |  |  |  |  | 0.9527 |
| 3 |  | 60 | 0.8 |  |  |  |  |
| 4 |  | 60 | 3.0 |  |  |  |  |
| 5 | SV |  | 3.0 |  |  |  |  |

For answers that require calculations, show your work below as you feel necessary. Full credit will be given if we only have to punch your numbers into a calculator.

Problem 2 (34 points)
A saturated mixture of water with an initial quality of $50 \%$, a pressure of 100 kPa , and mass of 2 kilograms is contained in a piston cylinder device. During a constant pressure process, the water is heated to a temperature of $200^{\circ} \mathrm{C}$.
(a) Sketch the process on a $P-v$ and a $T-v$ diagram. Clearly label the initial state, final state, and the process line.
(b) Determine the magnitude and direction of the work and heat transfer for
 the process, in kJ .

Problem 3 (34 points)
One way to provide air for cooling the passengers on an airliner is to expand high pressure air through an adiabatic, steady-state turbine as shown in the figure.
The inlet and outlet information is shown on the figure. The inlet volumetric flow rate is $0.2 \mathrm{~m}^{3} / \mathrm{s}$, and the turbine has an isentropic efficiency of 75\%.

Assume air can be modeled as an ideal gas and use the ideal gas tables. Do not interpolate-use closest values.
(a) Sketch the process on a $T-s$ diagram.
(b) Determine the power output from the turbine, in kW .
(c) Determine the outlet air temperature.


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